

Letters to the Editor

On the Behavior of Circulation Ratio and Algebraic Slip

In calculations of flow distribution within steam generators, an increase of circulation ratio with increasing algebraic slip has been reported. For example, in Ref. 1 a homogeneous calculation at full power produced a circulation ratio of 3.38, while a calculation for the same conditions employing an algebraic slip model using parameters developed by Lellouche and Zolotar² yielded a value of 3.63. Since the inclusion of inter-phase slip would allow the vapor phase to travel upward faster than the liquid, slip would intuitively be expected to reduce rather than raise the circulation ratio.

Though this expectation is generally correct, conditions exist for which the reverse is true, as will be shown in this Letter. This may be seen by considering a simple model of the upward flow through the evaporator of a recirculating steam generator. This flow is driven by the effective density difference between the liquid in the downcomer and the average mixture in the evaporator. Hence,

$$(\rho_l - \bar{\rho})gH = \frac{K\dot{m}^2}{2\bar{\rho}A^2}, \quad (1)$$

where

K = equivalent loss factor for the total evaporator flow \dot{m}

$\bar{\rho}$ = average density of the mixture.

Since an increase in $\bar{\rho}$ with slip tends to decrease both the driving head on the left side of Eq. (1) and the dynamic pressure loss on the right side, the behavior described below is a consequence of the relative importance of these two opposing effects. This equation may be rearranged to become

$$c(\bar{\rho}\rho_l - \bar{\rho}^2)^{1/2} = \dot{m}, \quad (2)$$

where c is a constant which lumps together all the constants of Eq. (1).

Since the circulation ratio, CR , is defined as the ratio of the total evaporator flow to the steam flow and the steam flow equals the feedwater flow in steady state (neglecting small carry-over and carryunder), Eq. (2) may be rewritten as

$$\frac{c(\bar{\rho}\rho_l - \bar{\rho}^2)^{1/2}}{\dot{m}_f} = CR, \quad (3)$$

where \dot{m}_f is the feedwater flow rate.

Differentiating this with respect to $\bar{\rho}$, the only variable for a given set of operating conditions, we obtain

$$\frac{d(CR)}{d\bar{\rho}} = \frac{c}{\dot{m}_f} \frac{(\rho_l - 2\bar{\rho})}{2(\bar{\rho}\rho_l - \bar{\rho}^2)^{1/2}}. \quad (4)$$

Thus, if ρ_l is greater than twice the average density of the flow, CR will increase with increasing $\bar{\rho}$. Since $\bar{\rho}$ increases with slip, CR will also increase with slip under the same circumstance. However, the increase of $\bar{\rho}$ with slip will eventually cause the right side of Eq. (4) to become negative. Beyond that point, additional slip will reduce the circulation ratio. This result is illustrated in Fig. 1 (from Ref. 3), where the circulation ratio calculated by a development version of the ATHOS code is plotted as a function of assumed slip velocity. Note the denominator of Eq. (4) must be real for any circulating flow.

By way of illustration, consider the behavior of $\rho_l - 2\bar{\rho}$ at 1000 psia for a variety of slip ratios and an assumed circulation ratio of 3.5. The average flow quality was assumed to be half the exit flow quality, i.e., $\frac{1}{2} \times \frac{1}{3.5} = 0.143$. The variation of $\rho_l - 2\bar{\rho}$ with slip ratio is shown in Table I.

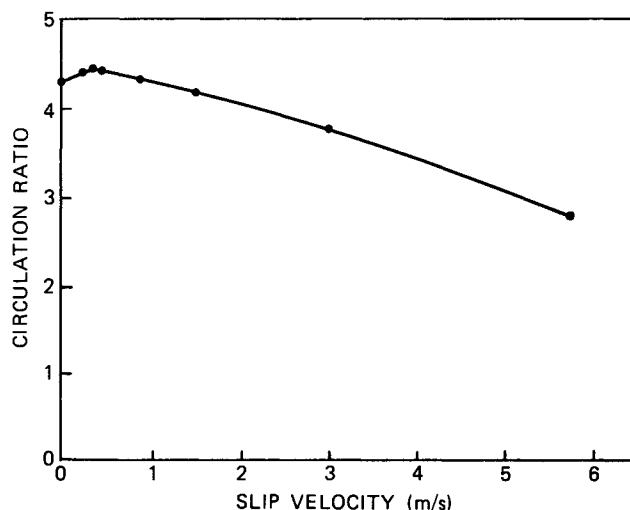


Fig. 1. Circulation ratio as a function of assumed slip velocity.

TABLE I
Variation of $\rho_l - 2\bar{\rho}$ with Slip Ratio

	Slip Ratio					
	1.0	1.5	2.0	3.0	4.0	5.0
$\rho_l - 2\bar{\rho}$	22	15.1	9.44	0.79	-5.5	-10.4

For an average slip ratio <3.109 , $\rho_l - 2\bar{\rho}$ is positive, so increasing slip increases the circulation ratio. Since the conditions assumed here are similar to those from Ref. 1 cited above, the behavior reported there is physically reasonable.

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REFERENCES

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2. G. S. LELLOUCHE and B. A. ZOLOTAR, "A Mechanistic Model for Predicting Two-Phase Void Fraction for Water in Vertical Tubes, Channels and Rod Bundles," EPRI NP-2246-SR, Electric Power Research Institute (1982).
3. L. W. KEETON et al., "The URSULA2 Computer Program, Vol. 2: Applications (Sensitivity Studies and Demonstration Calculations)," EPRI NP-1315, Electric Power Research Institute (Jan. 1983). Used by permission.

Richardson Extrapolation

The only aim of the present Letter is to draw the readers' attention to an old method¹ for increasing the accuracy of numerical solutions of linear problems. Customarily, new finite element (FE) or coarse-mesh (CM) algorithms are checked on well-defined benchmark problems,² and the reference solution of a given benchmark problem is usually obtained by a well-established program using a large number of meshes.

We are concerned here with the Richardson extrapolation,³ which allows one to obtain higher accuracy without refining further the mesh. The method can be used independently of the geometry.

It is well known that the accuracy of a numerical solution is often proportional to some power of the mesh size h . The accuracy of the finite difference (FD) method is $O(h)$, that of the linear FE method is $O(h^2)$. The basic idea in Richardson extrapolation is to separate a part of the error, proportional to some power of h , and to eliminate it. Let us consider the problem to be solved as

$$Lu = f \quad \text{in } \Omega, \quad (1a)$$

$$Bu = g \quad \text{on } \partial\Omega, \quad (1b)$$

where $\partial\Omega$ is the boundary of the region Ω . In numerical methods Eq. (1) is substituted by the discretized formulas

$$L_h u_h = f_h \quad \text{in } \Omega_h \quad (2a)$$

$$B_h u_h = g_h \quad \text{on } \partial\Omega_h. \quad (2b)$$

In the discretized formulas the dependence on the mesh size h is indicated explicitly. The discretized form of the most fre-

quently used operators is available in handbooks (see Ref. 4). Let us assume Eq. (2) to have a unique solution that is sufficiently smooth.³ The solution of Eq. (2) then has the following form:

TABLE I

Richardson Extrapolation of BUG-180 Results, Using 3 and 12 Points per Hexagon, Problem GA9A1

Number	B3 ^a	B12 ^a	RE ^b	B48 ^a
1	0.3798	0.3755	0.3741	0.3745
2	0.9988	1.0497	1.0667	1.0655
3	0.7781	0.8207	0.8349	0.8338
4	0.1891	1.2247	1.2366	1.2349
5	1.2358	1.2777	1.2917	1.2902
6	1.2183	1.2662	1.2821	1.2806
7	1.2450	1.2906	1.3058	1.3043
8	1.1999	1.2404	1.2539	1.2522
9	1.2147	1.2470	1.2578	1.2562
10	0.7442	0.7638	0.7703	0.7695
11	1.1766	1.1692	1.1667	1.1665
12	1.1328	1.1431	1.1465	1.1455
13	1.1938	1.2149	1.2219	1.2207
14	1.1565	1.1761	1.1826	1.1812
15	1.1522	1.1551	1.1561	1.1552
16	1.1802	1.1699	1.1665	1.1663
17	0.3494	0.3374	0.3334	0.3339
18	0.8565	0.8673	0.8709	0.8703
19	0.9093	0.9284	0.9348	0.9338
20	0.9547	0.9879	0.9990	0.9977
21	0.9665	1.0048	1.0176	1.0162
22	0.9419	0.9702	0.9796	0.9785
23	0.8859	0.9007	0.9056	0.9048
24	0.7534	0.7280	0.7195	0.7206
25	1.0347	0.9833	0.9662	0.9688
26	1.1301	1.1012	1.0916	1.0928
27	1.1038	1.0702	1.0590	1.0605
28	0.9614	0.9063	0.8879	0.8907
29	0.6444	0.6412	0.6401	0.6404
30	0.9144	0.8673	0.8576	0.8541
31	0.9867	0.9557	0.9454	0.9467
32	1.0122	1.0052	1.0029	1.0028
33	1.0913	1.0852	1.0832	1.0830
34	1.1010	1.0745	1.0657	1.0668
35	0.9778	0.9306	0.9149	0.9174
36	0.6438	0.6409	0.6399	0.6402
37	0.9128	0.8654	0.8496	0.8520
38	1.0073	0.9696	0.9570	0.9587
39	1.1131	1.0972	1.0919	1.0921
40	1.0477	1.0454	1.0446	1.0441
41	1.0215	1.0056	1.0003	1.0008
42	0.9655	0.9256	0.9123	0.9141
Time ^c	1.92	13.27	15.19	160.9

^aBn: BUG-180 result using n points per hexagon.

^bRE: Richardson extrapolation.

^cUNIVAC 1108 CPU min.